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- [001] KICK-DOWN SWITCHING SPEED OPTIMIZATION FOR AN AUTOMATIC TRANSMISSION OF A MOTOR VEHICLE
- [002]
- [003] The invention concerns a method for kick-down switching speed optimization in a motor vehicle with an automatic transmission, according to the pre-characterizing portion of Claim 1.
- [004]
- [005] For changing gear in an automatic transmission a certain delay time is allowed for before the switching is completed. During this, the engine speed increases until the load is taken up by the coupling being engaged from the coupling being disengaged. The reasons for the delay time are clutch filling times and ramp times until the switching pressure required for the change has been built up.
- [006] In vehicles with powerful engines the engine speed difference that occurs, having regard to the clutch filling times including the load take-up, can be as much as about 1200 r/min.
- [007] The maximum attainable engine speed in kick-down upshift gear changes can therefore assume various values which are affected by the load condition of the vehicle and the road inclination.
- [008] The problem arises that in an unladen vehicle moving downhill, kick-down (KD) upshifts take place in the range of the engine cut-off speed, i.e. the maximum permissible engine speed. The reason for this is the negative driving resistance, by which the vehicle is additionally accelerated.
- [009] In such a case the kick-down upshift must take place and adjusted earlier, i.e. at lower output speed. In the loaded condition and driving uphill, in contrast, the KD switching speed is lower as a result.
- [010] Accordingly, the optimum condition, namely equal KD switching speed for any kick-down upshift, cannot be achieved.
- [011] In the prior art kick-down shifts are triggered when predetermined output speeds are exceeded. These output speed thresholds can be stored in switching programs or be defined as discrete parameters.

- [012] The purpose of the present invention, starting from the aforesaid prior art, is therefore to provide a method for kick-down switching speed optimization in a motor vehicle with an automatic transmission, such that in kick-down upshifts the engine is prevented from exceeding the maximum permissible speed. In addition, premature upshifting is to be prevented.
- [013] According to the invention this objective is achieved by the characteristics of Claim 1. Other features of the invention emerge from the subordinate claims.
- [014]
- [015] Accordingly, it is proposed to determine the kick-down upshift point adaptively, i.e. as a function of the respective load conditions and road inclinations, so that switching takes place at a desired maximum engine speed.
- [016] According to the invention, when a kick-down condition is detected by the transmission control system, a speed offset  $nd\_abkd$  is added to the current upshift point. This speed offset is of appropriate sign and is stored in the transmission control system in the form of a characteristic line, a separate characteristic line being stored for each upshift.
- [017] According to a variant of this invention, when a kick-down condition has been recognized the target gear of the next upshift and the transmission output speed gradient are determined.
- [018] Thereafter, the speed offset  $nd\_abkd$  is calculated. For this purpose the delay times for the individual gear shifts are stored for application.
- [019] In an advantageous variant, the value of the speed offset is determined in the form of a characteristic line in accordance with both of the above methods and then recalculated as a function of the driver's activity (for example, by means of a valuation counter), so that  $n\_abkd$  will be higher in a KD-upshift by a sporty driver than in the case of a more sparing driver.
- [020] According to this variant, the characteristic line is multiplied by a factor that characterizes the driver's activity as a function of the gear change and output speed gradient. In this case the characteristic line always gives positive values. Alternatively, different characteristic lines are established for various characteristic

driver behaviors (again as a function of the gear change and output speed gradient). By averaging between the driver types, intermediate types of drivers can be allowed for.

[021]

[022] Below, the invention is explained in greater detail with reference to the figures, which show:

[023] Fig. 1 is a time-engine speed ( $n_{mot-t}$ ) diagram, which illustrates the problem upon which the invention is based, and

[024] Fig. 2 is a representation of the speed offset as a function of the output speed gradient according to the invention

[025]

[026] As illustrated in Fig. 1, when a gear shift is triggered by reaching a certain predetermined switching speed, the delay time  $\Delta t$  for the gear change can be longer than the time remaining before the maximum permissible engine speed is reached. This is particularly the case when driving downhill, when because of the negative road inclination the vehicle is accelerated additionally. This causes the maximum engine speed to be exceeded and is represented by drive A in Fig. 1. In contrast, an optimum kick-down upshift is illustrated by curve B. In the latter case the shift takes place at the maximum engine speed but without exceeding it.

[027] On the example of a gear upshift, Fig. 2 shows the engine speed offset according to the invention as a function of the output speed gradient  $ng_{ab}$ .

[028] When a kick-down condition is recognized by the transmission control system, a speed offset  $nd_{abkd}$  of appropriate sign is added to the current upshift point.

[029] On the example of the upswitch shown in Fig. 2, the value of the speed offset  $nd_{abkd}$  is equal to zero at a certain output speed gradient  $ng_{ab}$  (here for example 450 r/min per second). At a higher  $ng_{ab}$  value (for example, when driving downhill) the speed offset  $nd_{abkd}$  according to the invention will assume a negative value, i.e. the gear change will be triggered at a lower output speed.

In the case when the output speed gradient is smaller than a predetermined value, the speed offset  $nd\_abkd$  is positive, i.e. the gear change takes place at a higher output speed value.

- [030] The variation of the speed offset of appropriate sign is stored in the transmission control system in the form of a characteristic line, and for each upshift X a separate characteristic line  $KL\_ND\_ABKDX$  is stored:

$$nd\_abkdX = KL\_ND\_ABKDX [ng\_ab]$$

- [031] By virtue of this procedure the kick-down upshift point is made adaptive, i.e. it is determined as a function of the respective load conditions and road inclinations, so that the gear change takes place at a desired maximum engine speed.

- [032] Alternatively, in a variant of the present invention, instead of a speed offset an absolute kick-down switching characteristic line can be used.

- [033] According to a further variant of this invention, when a kick-down condition has been recognized the target gear of the next upshift and the transmission output speed gradient are determined.

- [034] Then, the speed offset  $nd\_abkd$  is calculated. For this purpose, the delay times for the individual gear changes are stored for application. This procedure has the advantage that temperature-dependent delay times are taken into account when calculating the speed offset  $nd\_abkd$ , as indicated by the following equation which shows an example of a possible calculation process:

$nd\_abkd = ng\_ab * KW\_TD\_KDX [CGT]$  with CGT as the transmission temperature and  $KW\_TD\_KD$  as the characteristic line of delay times for individual gear changes.

- [035] For the adaptive kick-down upshift speed for a particular upshift X, this gives:

$$n\_abkdX = KW\_ND\_ABKDX - nd\_abkd.$$